

**Warsaw University
of Technology**



**Faculty of Power and
Aeronautical Engineering**

WARSAW UNIVERSITY OF TECHNOLOGY

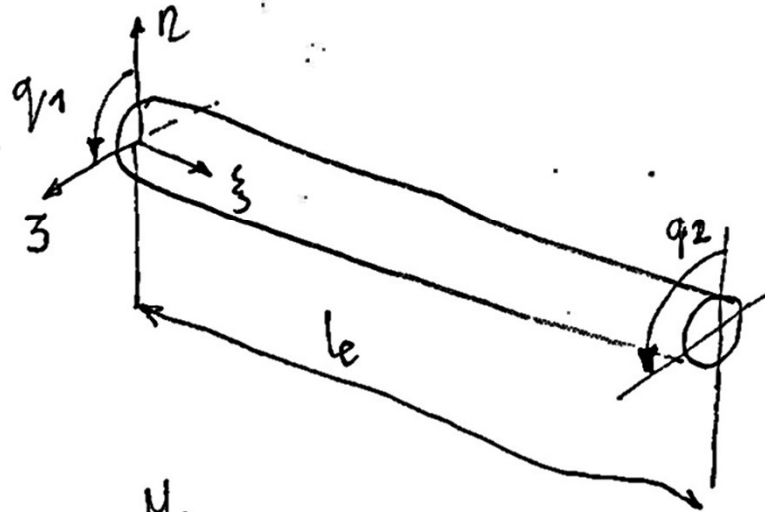
Institute of Aeronautics and Applied Mechanics

Finite element method (FEM)

Torsion bar finite element

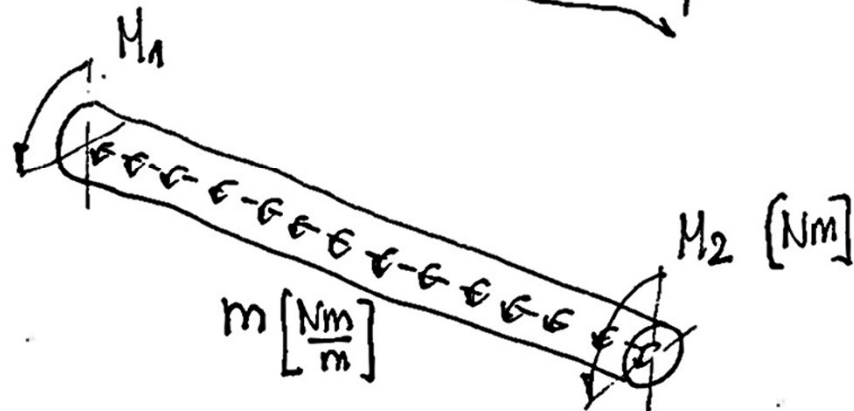
04.2021

TORSION BAR



q_1, q_2 - rotations
(twist angles)

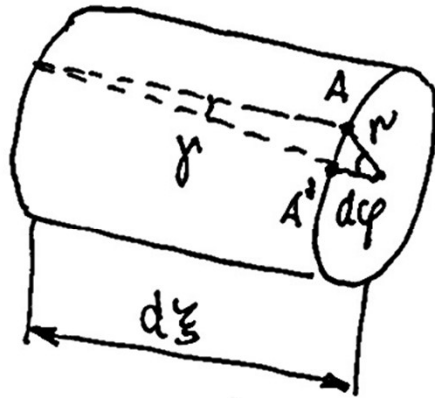
load



M - torque

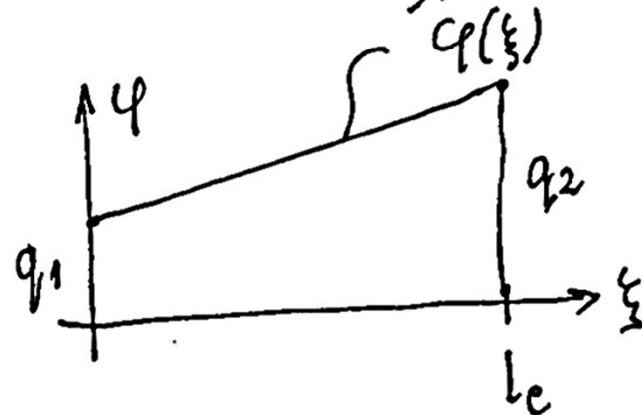
torque per unit length

deformation



$$AA' \cong \gamma \cdot d\xi = r d\varphi$$

$$\gamma = r \frac{d\varphi}{d\xi}$$



$$N_1(\xi) = 1 - \frac{\xi}{l_e}$$

$$N_2(\xi) = \frac{\xi}{l_e}$$

$$\varphi(\xi) = N_1(\xi) \cdot q_1 + N_2(\xi) \cdot q_2 =$$

$$= \begin{bmatrix} N_1 \\ N_2 \end{bmatrix} \cdot \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix}$$

$$\frac{d\varphi}{d\xi} = \begin{bmatrix} \frac{dN_1}{d\xi} & \frac{dN_2}{d\xi} \end{bmatrix} \cdot \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} = \begin{bmatrix} -\frac{1}{l_e} & \frac{1}{l_e} \end{bmatrix} \cdot \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix}$$

elastic strain energy

$$U_e = \frac{1}{2} \int_{\Omega_e} \boldsymbol{\tau} \cdot \boldsymbol{\gamma} \, d\Omega_e = \frac{1}{2} \int_{\Omega_e} G \cdot \gamma^2 \, d\Omega_e = \frac{1}{2} \int_{\Omega_e} G r^2 \left(\frac{d\varphi}{d\xi} \right)^2 d\Omega_e =$$

Hook's law

$$\boldsymbol{\tau} = G \cdot \boldsymbol{\gamma} = \frac{E}{2(1+\nu)} \cdot \boldsymbol{\gamma}$$

$$= \frac{1}{2} \int_0^l G \frac{d\varphi}{d\xi} \cdot \frac{d\varphi}{d\xi} \cdot \underbrace{\int_A r^2 dA}_{J_s} d\xi =$$

polar moment of inertia J_s

$$J_s = \frac{\pi d^4}{32} \text{ for a circle}$$

$$= \frac{1}{2} \int_0^l GJ_s \cdot \frac{d\varphi}{d\xi} \cdot \frac{d\varphi}{d\xi} d\xi =$$

$$\begin{matrix} \uparrow & & \uparrow \\ Lq \int_e & \left\{ \begin{matrix} \frac{dN_1}{d\xi} \\ \frac{dN_2}{d\xi} \end{matrix} \right\} & \left[\begin{matrix} \frac{dN_1}{d\xi} & \frac{dN_2}{d\xi} \end{matrix} \right] \cdot \left\{ \begin{matrix} q \\ \end{matrix} \right\}_e \\ 1 \times 2 & & 2 \times 1 \end{matrix}$$

$$= \frac{1}{2} Lq \int_e \cdot [k]_e \cdot \left\{ q \right\}_e ; \text{ where:}$$

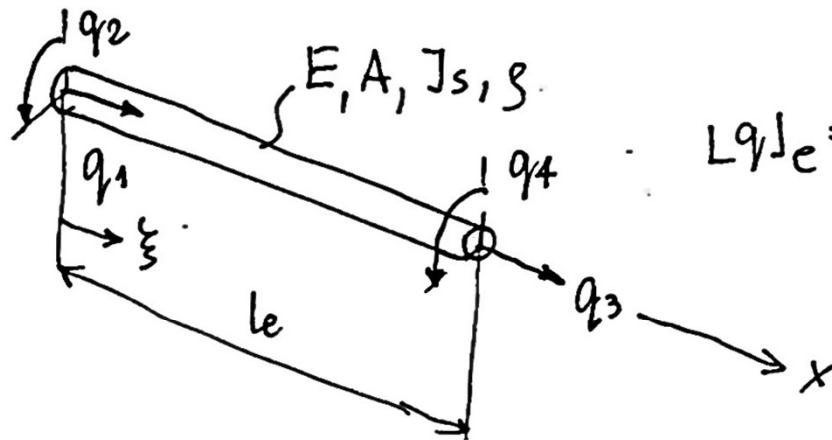
$$[k]_e = GJ_s \begin{bmatrix} \int_0^l \frac{dN_1}{d\xi} \frac{dN_1}{d\xi} d\xi & \int_0^l \frac{dN_1}{d\xi} \frac{dN_2}{d\xi} d\xi \\ \int_0^l \frac{dN_2}{d\xi} \frac{dN_1}{d\xi} d\xi & \int_0^l \frac{dN_2}{d\xi} \frac{dN_2}{d\xi} d\xi \end{bmatrix} = \frac{GJ_s}{l} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

potential energy of loading

$$W_e = \underset{1 \times 2}{[F]}_e \cdot \{q\}_e + M_1 \cdot q_1 + M_2 \cdot q_2$$

$$\underset{1 \times 2}{[F]}_e = L \left[\int_0^L m(\xi) \cdot N_1(\xi) d\xi, \int_0^L m(\xi) \cdot N_2(\xi) d\xi \right]$$

AXIAL + TORSION BAR



$$[q]_e = [q_1, q_2, q_3, q_4]$$

AXIAL BAR:

$$[k]_{2 \times 2, A} = \frac{EA}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$[q]_A = [q_1, q_3] \text{ (translational D.O.F)}$$

$$[k]_{4 \times 4, A}^* = \frac{EA}{l_e} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

TORSION BAR:

$$[k]_{2 \times 2, T} = \frac{GJ_s}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$[q]_T = [q_2, q_4] \text{ (rotational D.O.F)}$$

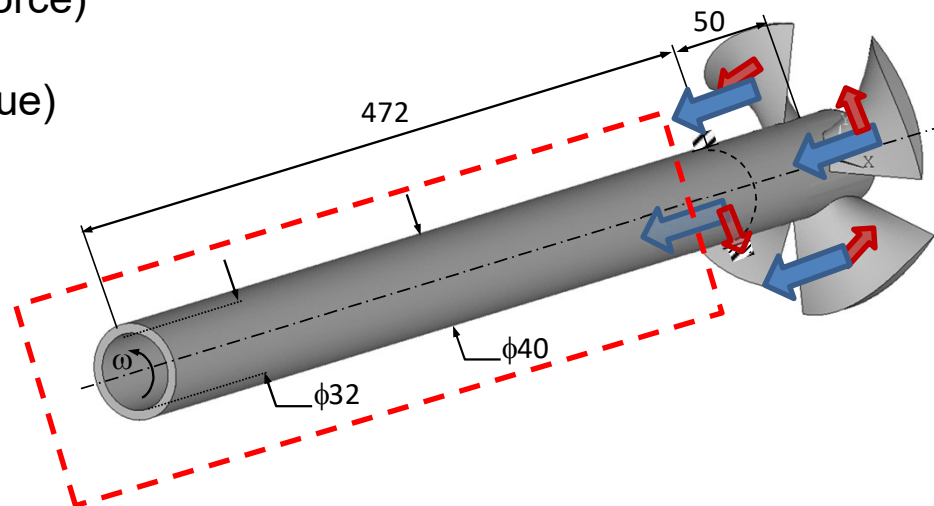
$$[k]_{4 \times 4, T}^* = \frac{GJ_s}{l_e} \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix}$$

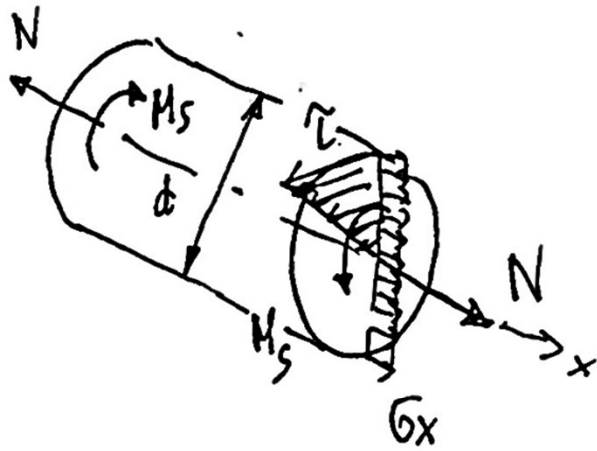
$$[k]_e = [k]_A^* + [k]_T^* = \frac{1}{l_e} \begin{bmatrix} EA & 0 & -EA & 0 \\ 0 & GJ_s & 0 & -GJ_s \\ -EA & 0 & EA & 0 \\ 0 & -GJ_s & 0 & GJ_s \end{bmatrix}$$

Example. Axial+torsion element used to model the propeller shaft

← lift force (axial force)

← drag force (torque)





$$\begin{aligned} \sigma_x &= E \cdot \epsilon_x = E \cdot \frac{du}{d\xi} = \\ &= E \left[\frac{dN_1}{d\xi}, \frac{dN_2}{d\xi} \right] \cdot \begin{Bmatrix} q_1 \\ q_3 \end{Bmatrix}_A = \\ &= E \frac{q_3 - q_1}{l} \end{aligned}$$

$$\frac{dN_1}{d\xi} = -\frac{1}{l} \quad , \quad \frac{dN_2}{d\xi} = \frac{1}{l}$$

$$\begin{aligned} \tau(r) &= G \cdot \gamma(r) = G \cdot r \frac{d\varphi}{d\xi} = G \cdot r \left[\frac{dN_1}{d\xi}, \frac{dN_2}{d\xi} \right] \cdot \begin{Bmatrix} q_2 \\ q_4 \end{Bmatrix}_T = \\ &= G \cdot \frac{q_4 - q_2}{l} \cdot r \quad , \quad \tau_{\max} = \frac{Gd}{2} \cdot \frac{q_4 - q_2}{l} \end{aligned}$$

$$\sigma_{\text{eqv}}^{\text{max}} = \sqrt{\sigma_x^2 + 3\tau_{\max}^2}$$

